

# 9210-130 Level 6 Graduate Diploma in Engineering

Mechanics of machines and strength of materials

# Sample Paper

# You should have the following for this examination

- one answer book
- non-programmable calculator
- pen, pencil, ruler, drawing
- instruments

# The following data is attached

- Worksheet booklet WSQ2
- Worksheet booklet WSQ5

# **General instructions**

- This examination paper is of **three hours** duration.
- This examination paper contains nine questions in sections A and B.
- Answer **five** questions selecting at least **two** questions from each section.
- All the question carry equal marks. The maximum marks for each section within a question are given against that section.
- An electronic, non-programmable calculator may be used, but the candidates **must** show clearly the steps prior to obtaining final numerical values.
- Drawing should be clear, in good proportion and in pencil. Do **not** use red ink.

# Section A

b)

1 a) A rod of length L, cross sectional area  $A_1$  and modulus of elasticity  $E_1$  has been placed inside a tube of the same length L but of cross sectional area  $A_2$  and modulus of elasticity  $E_2$  as shown in Figure Q1(a).



#### Figure Q1(a)

i) What is the deformation of the rod and tube when a force P is exerted on a rigid end plate as shown? (6 marks) ii) If the axial forces in the rod and the tube are P<sub>1</sub> and P<sub>2</sub> respectively, show that  $P_1 = \frac{A_1E_1P}{A_1E_1 + A_2E_2}$  and  $P_2 = \frac{A_2E_2P}{A_1E_1 + A_2E_2}$  (6 marks) As shown in Figure Q1(b), a 1.35 m long and 0.50 m diameter concrete post is reinforced with six steel bars, each with of 30 mm diameter. If the modulus of rigidity of steel E<sub>s</sub> = 200 GPa and modulus of rigidity of concrete E<sub>c</sub> = 30 GPa. Determine the normal stresses in the steel and in the concrete when a 1500 kN axial centric force P is applied to the post. (8 marks)



Figure Q1(b)

2 a) Derive an expression for the stresses on an oblique plane of a rectangular body, when the body is subjected to a simple shear stress.

(6 marks)

(4 marks)

(6 marks)

b) At a point in a strained material, on plane BC there are normal and shear stresses of 640 N/mm<sup>2</sup> and 160 N/mm<sup>2</sup> respectively. On plane AC, which is perpendicular to BC, there are normal and shear stresses of 320 N/mm<sup>2</sup> and 160 N/mm<sup>2</sup> respectively. These stresses are shown in Figure Q2. Determine the followings.



# Figure Q2

- i) Principal stresses and location of the planes on which they act. (10 marks)
- ii) Maximum shear stress and the plane on which it acts.

# 3 a) Show that $I_o = I_G + Ah^2$ ,

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where  $I_G$  is the second moment of area of a cross section with area 'A' about an axis through the centroid and 'h' is the distance from the centroid to a parallel axis in the same plane about which its second moment of area is  $I_o$ .

b) Considering section Y-Y of the beam and loading shown in Figure Q3, determine



# Figure Q3

|          | <ul><li>i) The largest shearing stress in that section.</li><li>ii) The shearing stress at point 'R'.</li></ul>   | (7 marks)<br>(7 marks) |
|----------|---|------------------------|
| a)       | Explain the meaning of each symbol in the simple torsion formula given by   |                        |
|          | $\frac{1}{J} = \frac{\tau}{r} = G \frac{\Theta}{I}$   | (4 marks)              |
| b)<br>c) | Write down <b>four</b> assumptions made to derive the torsion formula.<br>A solid circular shaft transmits 60 kW power at 120 rev/min. Calculate the<br>shaft diameter, if the twist in the shaft is not to exceed 1° in 3 m length of<br>shaft, and shear stress is limited to 50 MPa. Take Modulus of Rigidity of the | (4 marks)              |
|          | material $G = 80 GPa$ .   | (12 marks)             |

# Section **B**

5 a) Figure Q5(a) shows a slider A moving outwards on the rod OB at a velocity v and an acceleration  $\boldsymbol{a}$ . The rod has an angular velocity  $\boldsymbol{\omega}$  counterclockwise about O and an acceleration  $\boldsymbol{\alpha}$  clockwise as shown in the figure. OA = r. Determine the absolute acceleration of the slider A.

(6 marks)



Figure Q5(a)

b) Figure Q5(b) shows a scotch yoke mechanism in which the slider P moves along the guide path of yoke R. At the instant shown in the figure, the crank OP makes an angle of 30° with the horizontal and its angular velocity and angular acceleration are 10 rad/s and 20 rad/s<sup>2</sup> respectively. OP = 300 mm. Determine



- i) The horizontal velocity of the yoke.
- ii) The acceleration of the slider P in the yoke.
- iii) The horizontal acceleration of the yoke.

(6 marks) (4 marks) (4 marks)

|          | at a linear speed v  |   |
|----------|--|---|
|          | i) Show that the centrifugal tension in the belt ( $T_c$ ) is given by $T_c = mv^2$  | (3 marks)   |
|          | ii) Show that, $\frac{T_1 - T_c}{T_2 - T_c} = e^{\mu \theta}$ , where $T_1$ and $T_2$ are the tensions in the tight side and   |   |
| b)       | <ul> <li>slack side respectively in an open belt drive where μ is the coefficient of friction between the belt and the pulley and θ is the angle of lap.</li> <li>Power is transmitted between two parallel shafts, the axes of which are 2 m apart, by means of an open belt of thickness 3 mm and width 60 mm. The diameter of the pulley on the driving shaft is 800 mm and it rotates at 360 rev/min. The speed ratio between the driving and driven shafts is 1.8. The belt material has a density of 3 kg/m and a permissible stress of 5 N/mm<sup>2</sup>. The coefficient of friction between the belt and the pulley is 0.25.</li> <li>Determine</li> </ul> | (5 marks)   |
|          | <ul><li>i) The angle of lap on the smaller pulley.</li><li>ii) The maximum power transmitted.</li></ul>  | (4 marks)<br>(8 marks)  |
| a)<br>b) | Explain the terms, sensitiveness, hunting, and stability relating to governors.<br>In a spring controlled Hartnell governor, the length of the sleeve arm is 150 mm<br>and that of the ball arm is 125 mm. The mass of each ball is 3 kg and the fulcrum<br>of each bell crank lever is at a distance of 160 mm from the axis of rotation of the<br>governor. The extreme radii of the balls are 130 mm and 170 mm. The maximum<br>equilibrium speed is 10% higher than the minimum equilibrium speed and the<br>stiffness of the control spring is 5 N/mm.<br>Neglecting obliguity of arms and friction determine   | (6 marks)   |
|          | <ul><li>i) The maximum and minimum equilibrium speeds of the governor.</li><li>ii) The equilibrium speed when the ball arms are vertical.</li></ul>  | (8 marks)<br>(6 marks)  |
| a)<br>b) | Explain why it is necessary to attach a flywheel to a machine, which requires<br>a non-uniform operating torque when it is driven by a constant torque motor.<br>A constant torque motor of capacity 2.5 kW drives a riveting machine.<br>The moving parts including the flywheel are equivalent to a mass of 120 kg<br>at a radius of gyration of 750 mm. Each riveting operation absorbs 10 kJ of energy<br>and the operation takes 1.2 seconds. The speed of the flywheel just before the<br>riveting operation is 300 rev/min. Determine   | (4 marks)   |
|          | <ul> <li>i) The speed of the flywheel immediately after riveting operation.</li> <li>ii) The number of rivets completed per minute.</li> </ul>   | (12 marks)<br>(4 marks)   |
|          | b)<br>a)<br>b)   | <ul> <li>at a linear speed v.</li> <li>i) Show that the centrifugal tension in the belt (<i>T<sub>c</sub></i>) is given by <i>T<sub>c</sub></i> = <i>mv<sup>2</sup></i></li> <li>ii) Show that, <i>T<sub>1</sub></i> - <i>T<sub>c</sub></i> = <i>e<sup>µθ</sup></i>, where <i>T</i><sub>1</sub> and <i>T</i><sub>2</sub> are the tensions in the tight side and slack side respectively in an open belt drive where µ is the coefficient of friction between the belt and the pulley and θ is the angle of lap.</li> <li>b) Power is transmitted between two parallel shafts, the axes of which are 2 m apart, by means of an open belt of thickness 3 mm and width 60 mm. The diameter of the pulley on the driving shaft is 800 mm and it rotates at 360 rev/min. The speed ratio between the driving and driven shafts is 1.8. The belt material has a density of 3 kg/m and a permissible stress of 5 N/mm<sup>2</sup>. The coefficient of friction between the belt and the pulley is 0.25. Determine <ol> <li>The angle of lap on the smaller pulley.</li> <li>The angle of lap on the smaller pulley.</li> <li>The angle of lap on the smaller pulley.</li> <li>The maximum power transmitted.</li> </ol> </li> <li>a) Explain the terms, sensitiveness, hunting, and stability relating to governors.</li> <li>In a spring controlled Hartnell governor, the length of the sleeve arm is 150 mm and that of the ball arm is 125 mm. The mass of each ball is 3 kg and the fulcrum of each bell crank lever is at a distance of 160 mm from the axis of rotation of the governor. The extreme radii of the balls are 130 mm and 170 mm. The maximum equilibrium speed is 10% higher than the minimum equilibrium speed and the stiffness of the control spring is 5 N/mm. Neglecting obliquity of arms and friction determine <ol> <li>The enaximum and minimum equilibrium speeds of the governor.</li> <li>The maximum and minimum equilibrium speeds of the governor.</li> </ol> </li> <li>Aconstant torque motor of capacity 2.5 kW drives a riveting machine. The moving parts including the flywheel are equivalent to a mass of 120 kg at a radius of gyration of 750 mm. Each riveting operation absorbs 10 kl of energy and the operati</li></ul> |

9 a) Figure Q9 shows a uniform rod AB of length 3*a* and mass *m* hinged at A and supported by a spring of stiffness *k* at B. A mass *m* is attached to the rod at a distance *a* from A and a dashpot having a damping constant *b* is attached at a distance 2*a* from A.

Assuming the system to be under damped,

- i) Derive the equation of motion.
- ii) Obtain expressions for the damped natural frequency and the logarithmic decrement.

(4 marks)

(10 marks)

b) If m = 20 kg, k = 1000 N/m and b = 100 Ns/m, determine



#### Figure Q9

i) Frequency of damped vibrations. (3 marks)
 ii) The ratio of successive amplitudes on the same side of the equilibrium position. (3 marks)